# OPTIMIZATION OF MECHANICAL STRUCTURES USING INTERVAL ANALYSIS

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## 1 Introduction

The problem of optimal design consist in finding the optimum parameters "x" according to a prescribed optimality criterion

$$\begin{cases} \min J(\mathbf{x}) \\ p(\mathbf{x}) = 0, \ q(\mathbf{x}) \ge 0 \end{cases}$$
(1)

Existing optimization methods (solon.cma.univie.ac.at/~neum/glopt, plato.la.asu.edu/guide) usually aren't reliable or can't use the nondifferentiable and not continuous objective functions or constraints. An interval global optimization method [1,3] is: very stabile and robust, universally applicable and 100% reliable. The interval algorithm guarantees that all stationary global solutions have been found.

## 2 Interval global optimization

The set of all closed real intervals is denoted by I(R) and the natural interval extension of a real function f(x) is denoted by  $\hat{f}(.)$  [2]. The interval global optimization method is based on the properties of interval arithmetic [2]. If the following inequality holds

$$\hat{\mathbf{f}}([\mathbf{x}]_1) < \hat{\mathbf{f}}([\mathbf{x}]_2) \tag{2}$$

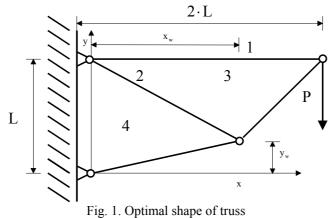
where  $[x]_1 = [x_1^-, x_1^+]$ ,  $[x]_2 = [x_2^-, x_2^+]$ ,  $\hat{f}:I(R) \rightarrow I(R)$ , then the global minimum is not in the interval  $[x]_2$ . Several techniques are used to improve the interval global optimization algorithm for example: a midpoint test, a monotonicity test, a concavity test, an interval Newton method, a parallelization, a local minimizer, a Fritz John condition and many others.

Today exist a lot of commercial and scientific software based on the interval global optimiation method (compare cs.utep.edu/interval-comp/intsoft). Many information about interval analysis can be found on the internet (cs.utep.edu/interval-comp/main).

In order to investigate the performance of the algorithms described above a test were performed for a truss shown in Fig. 70. Using interval global optimization we can find optimal shape of this construction. In calculation we assume that bar number 1 have a constant length, L=1 m,  $\sigma_0 = 190$  MPa and P=10 kN. The objective functions have the following form:

$$J = \frac{1}{\sigma_0} \sum_{i=1}^{4} |N_i| L_i \tag{3}$$

where N<sub>1</sub> are the axial forces. The optimal results are the following:  $x_w = 1.1895$  m,  $y_w = 0.223488$  m (compare Fig.1),  $A_1 = 5.493 \cdot 10^{-6}$  m<sup>2</sup>,  $A_2 = 6.011 \cdot 10^{-6}$  m<sup>2</sup>,



 $A_3 = 7.607 \cdot 10^{-6} \text{ m}^2$ ,  $A_4 = 1.071 \cdot 10^{-6} \text{ m}^2$ , where  $A_i$  are the cross sections of i-th bar.

#### References

- [1] HANSEN E., 1992, *Global Optimization Using Interval Analysis*, New York, Marcel Dekker
- [2] MOORE R.E., 1966, Interval Analysis, New York, Prentice-Hall, Englewood Cliffs
- [3] RATSCHEK H., ROKNE J., 1988, New Computer Methods for Global Optimization, New York, John Wiley & Sons

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